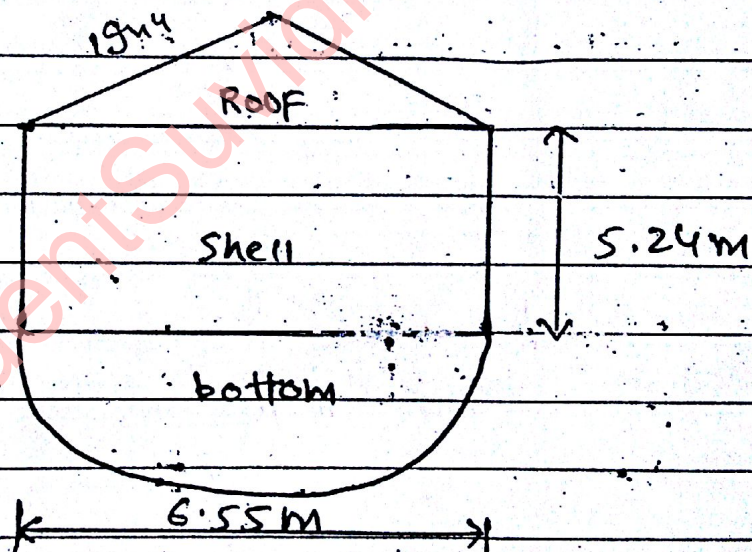


CIRCULAR TANK

Q1 Design a elevated, circular water Tank for a capacity of 2,50,000 litres. The height of the Tank bottom above the ground level is 8.7m. The tank is supported over 8 columns and is situated at Railway station in Allahabad.

Sol.



cylindrical Tank

(I) [Size of tank] :-

$$\begin{aligned}\text{Capacity of Tank} &= 2,50,000 \text{ l} \\ &= 250 \text{ m}^3\end{aligned}$$

Assume hemispherical bottom of the Tank
with conical roof (most common)

Assume height of cylindrical shell to be 0.8 times the Diameter of the Tank or $\left[\frac{H}{D} = 0.8 \right] \leftarrow \text{Fix}$

Now, $\text{Area} \cdot \text{Height} = \text{Volume}$

And $\text{Vol.} = 250 \text{ m}^3$

[Vol. of cylinder + Vol. of hemisphere = 250 m^3]

$$\pi r^2 H + \frac{1}{2} \times \frac{4}{3} \pi r^3 = 250$$

$$\frac{\pi D^2 H}{4} + \frac{1}{2} \times \frac{4}{3} \pi \left(\frac{D}{2} \right)^3 = 250$$

$$\therefore, \frac{\pi D^2 (0.8D)}{4} + \frac{1}{2} \times \frac{4}{3} \pi \left(\frac{D}{2} \right)^3 = 250$$

$$[D = 6.55 \text{ m}]$$

$$\text{Hence, } H = 0.8D = 0.8(6.55)$$

$$\therefore [H = 5.24 \text{ m}]$$

(II) [Thickness of plates] :-

Assume the efficiency of Joints to be 75% and let us provide 16 mm power driven Rivets for making connections.

→ Thickness of shell plates :-

$$[t = wHD]$$

$$t = \frac{w}{2} \times \frac{H}{0.75} \times \frac{D}{120}$$

$$t = \frac{9.81 \times 10^{-6}}{2} \times \frac{5.24 \times 10^3}{0.75} \times \frac{6.55 \times 10^3}{120}$$

$$[t = 1.87 \text{ mm}]$$

Here, w = unit wt. of water

i.e. 9.81 kN/m^3 or $9.81 \times 10^6 \text{ N/mm}^3$

Add 1.5 mm extra to account for corrosion.

$$t = 1.87 + 1.5 = 3.37 \text{ mm} < 6 \text{ mm}$$

\therefore provide 6 mm thick plates in the cylindrical shell of the Tank.

\rightarrow Thickness of suspended bottom plates :-

$$\left[t = \frac{w h D}{4 \eta \sigma_{at}} \right] \quad \left\{ h = H + \frac{D}{2} \right.$$

$$h = \frac{5.24 + 6.55}{2} = 8.515 \text{ m}$$

$$t = \frac{9.81 \times 10^{-6} \times 8.515 \times 10^3 \times 6.55 \times 10^3}{4 \times 0.75 \times 120}$$

$$[t = 1.52 \text{ mm}]$$

Add 1.5 mm extra for corrosion

$$t = 1.52 + 1.5 = 3.02 \text{ mm} < 6 \text{ mm}$$

\therefore Provide 6 mm thick plates in hemispherical bottom of tank.

Conical Roof :-

provide 5 mm thick plates for the Conical Roof. The pitch of the Roof may be kept 1 in 4.

III [Connections] :-

Provide 16 mm power Driven Rivets and double Riveted lap joints have been used all through.

→ cylindrical shell plates :-

Hoop stress per linear vertical height

$$[F_1 = \frac{wHD}{2}]$$

$$= \frac{9.81 \times 10^{-6} \times 5.24 \times 10^3 \times 6.55 \times 10^3}{2}$$

$$F_1 = [168.35 \text{ N/mm}]$$

Since Nominal Dia. = 16 mm

Gross Dia. = 16 + 1.5 = 17.5 mm

Strength of Rivets in single shear :-

$$= \frac{\pi}{4} \times (17.5)^2 \times \frac{(0.8 \times 90)}{1}$$

$$= \underline{17318.03 \text{ N}} \quad \left[\frac{0.8 \pi d^2 \tau_{VF}}{4 \cdot 1} \right]$$

Strength of Rivet in Bearing :-

$$= 17.5 \times 6 \times 0.8 \times 270 \quad [0.8 d t \sigma_{PF}]$$
$$= 22680 \text{ N} \quad \downarrow$$

(16mm) - 270

Rivet value, $R_v = 17318.03 \text{ N (Min.)}$

$$\frac{\text{Pitch of Rivets}}{F_1} = \frac{2 R_v}{F_1}$$

$$= \frac{2(17318.03)}{168.35} = 205.74 \text{ mm}$$

$\neq 10t$

Pitch $\neq 60 \text{ mm}$ $[10t, (10)(6), = 60]$

Rivets for horizontal joints are provided at the same pitch as that of vertical joints.

→ Hemispherical bottom plates :-

Load stress per unit length on the Radial Joints :-

$$\left[F_2 = \frac{w h D}{4} \right] \quad \left(h = H + \frac{D}{2} \right)$$

$$= \frac{9.81 \times 10^{-6} \times 8.515 \times 10^3 \times 6.55 \times 10^3}{4}$$

$$\left[F_2 = 136.78 \text{ N/mm} \right]$$

→ If it's not like this then repeat steps for rivet

Since $F_2 < F_1$. So, Rivets for making connections of hemispherical bottom

For Radial as well as horizontal joints

IV) Circular Girder :-

Weight of water, [W.V] $\rightarrow 10 \text{ kN/m}^3 (9.8)$

$$W_1 = 10 \times 250 = 2500 \text{ kN} \Rightarrow 2500 \times 10^3 \text{ N}$$

Self wt. of the Tank :-

$$W_2 = \left[\pi D H t + \frac{1}{2} 4 \pi r^2 t \right] \times 7.9 \times 10^{-5}$$
$$= \left[\pi (6.55) 10^3 \times (5.24 \times 10^3) \times 6 + \frac{1}{2} 4 \pi (6)$$

$$\left[\frac{6.55 \times 10^3}{2} \right]^2 \right] \times 7.9 \times 10^{-5}$$

Unit wt. of
Steel (79.5 kN/m^3)

$$\Rightarrow 83052.77 \text{ N}$$

Self wt. of Conical Roof :-

$$W_3 = \frac{\pi D}{2} \left[\sqrt{\left(\frac{D}{2}\right)^2 + \left(\frac{D}{4}\right)^2} \right] \times 7.9 \times 10^{-5}$$

$$\Rightarrow 14880.74 \text{ N}$$

The Self wt. of the Tank & Conical Roof may be increased by 20% to account additional wt. of overlap of plates & connectors.

$$W_4 = 1.2 (W_2 + W_3)$$
$$= 1.2 (83052.77 + 14880.74)$$

Assume self wt. of girder = 1500 N/m

$$\begin{aligned}\text{Total self wt. (girder)} &= 1500 \times (\pi \times 6.55) \\ &= 30866.14 \text{ N}\end{aligned}$$

$$\begin{aligned}\text{Now, Total load on girder :-} \\ &= 2500 \times 10^3 + 117520.21 + 30866.14 \\ &= [2650 \times 10^3 \text{ KN}]\end{aligned}$$

For circular girder supported on 8 columns :-

Fixed Formulas

$$(i) \text{ Max. B.M} = -0.00827 WR, (M)$$

(at supports)

$$= -0.00827 (2650 \times 10^3) \cdot \frac{6.55}{2}$$

$$= (-71173.26 \text{ N-m})$$

$$(ii) \text{ Max. Torsion} = 0.00063 WR, (T)$$

$$= 0.00063 (2650 \times 10^3) (6.55/2)$$

$$= (5467.6 \text{ N-m})$$

$$(iii) \text{ Max. S.F} = \frac{W}{16}, \quad (\text{For 8 columns})$$

$$= \frac{2650 \times 10^3}{16} = 165625.00 \text{ N}$$

Section Modulus Required :-

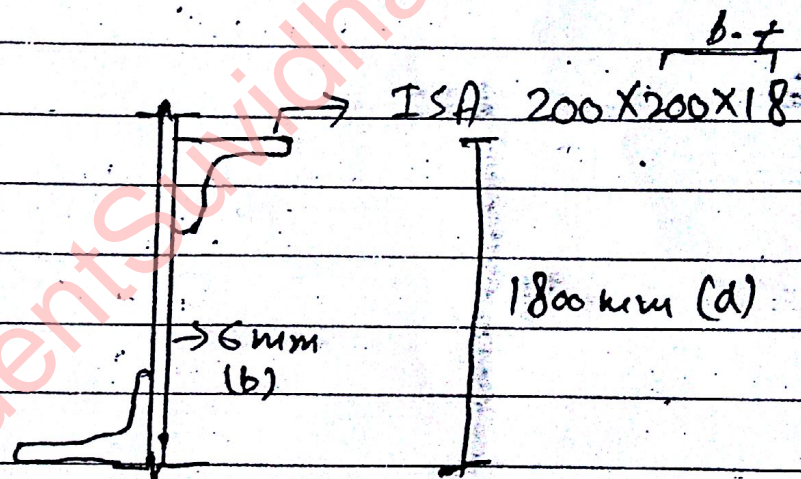
$$Z_{req} = \frac{\text{Max B.M}}{0.66 f_y} = \frac{M}{f_y}$$

$$= \frac{71773.26 \times 10^3}{0.66 (250)} = 0.66 f_y$$

$$= (434989.8 \text{ mm}^3)$$

Let us try girder section which is built up by using 18mm depth of cylindrical shell and 2 ISA 200 X 200 X 18 mm sections.

Choose ISA (equal or unequal angle) from code book, whose $Z > Z_{req}$. If not, then cal. I_x & Z_x if not available]



$$I_{xx} = \frac{6 \times 1800^3}{12} + 2 \times 2588.7 \times 10^4$$

$$+ 2 \times 6881 \times (900 - 56.1)^2$$

$$= 1276861.9 \times 10^4 \text{ mm}^4$$

$$Z_{prov} = \frac{1276861.9 \times 10^4}{900} \left(\frac{I}{y} \right)$$

$$= 14187354 > 434989.8 \text{ mm}^3$$

$$J = 2 \left[\frac{1}{3} b t^3 + (b-t) t^3 \right] + \frac{1}{3} d (t)^3$$

Torsion Constant :- $J = \sum \frac{1}{3} b t^3$

$$J = 2 \left[\frac{1}{3} (200)(18)^3 + (200-18)18^3 \right] + \frac{1}{3} (1800)(6)^3$$

$$= 3630048 \text{ mm}^4$$

shear stress due to Torsion :-

$$= \left[\frac{T}{J} \tau_{\max} \right]$$

$$= \frac{5467.6 \times 10^3 \times (18+6)}{3630048}$$

$$= (43.30 \text{ MPa})$$

shear stress due to S.F :-

$$= \frac{\text{Max. S.F}}{2 \times 6881 + 1800 \times 6}$$

$$\left[\frac{\text{Max S.F}}{2a + b.d} \right]$$

$$= (6.743 \text{ MPa})$$

Total shear stress :-

$$= 43.30 + 6.743$$

$$S = \underline{50.04 \text{ N/mm}^2} < 0.4 f_y$$

$$< 100 \text{ N/mm}^2$$

$$\Rightarrow (O.K)$$

$$\left[I_{xx} = \frac{1}{12} b d^3 + 2 I_{yy} + 2 a r^2 \right] \begin{matrix} a = \text{mm}^2 \\ I_{yy} = \text{mm}^4 \\ b = 6, d = 18 \end{matrix}$$

$$\text{And } \tau = (\gamma - \gamma_v)^2$$

Bending Stress, σ_{bct} :-

$$= \frac{M}{Z}$$

$$= \frac{71773.26 \times 10^3}{14187354} = 5.059 \text{ mpa}$$

Hoop stress, σ_{at1} :-

$$= \frac{wHD}{2rt}$$

$$= \frac{9.81 \times 10^{-6} (5.24 \times 10^3) (6.55 \times 10^3)}{2(0.75)(6)}$$

$$= 37.41 \text{ mpa}$$

Max. Principal stress :-

$$\sigma_1 = \frac{\sigma_{bct} + \sigma_{at1}}{2} + \sqrt{\left(\frac{\sigma_{bct} - \sigma_{at1}}{2}\right)^2 + (\tau)^2}$$

$$= \frac{37.41 + 5.059}{2} + \sqrt{\left(\frac{37.41 - 5.059}{2}\right)^2 + (5.059)^2}$$

$$= 75.59 \text{ mpa} < 0.6 f_y$$
$$< 0.6 (250)$$
$$< 165 \text{ mpa}$$

\Rightarrow (O.K)